

University of Naples "Federico II"
Department of Industrial Engineering
Fluid Power Research Group

NUMERICAL AND EXPERIMENTAL ANALYSES OF VANE AND PISTON PUMPS



**SCUOLA POLITECNICA E
DELLE SCIENZE DI BASE**

**DIPARTIMENTO DI
INGEGNERIA
INDUSTRIALE**



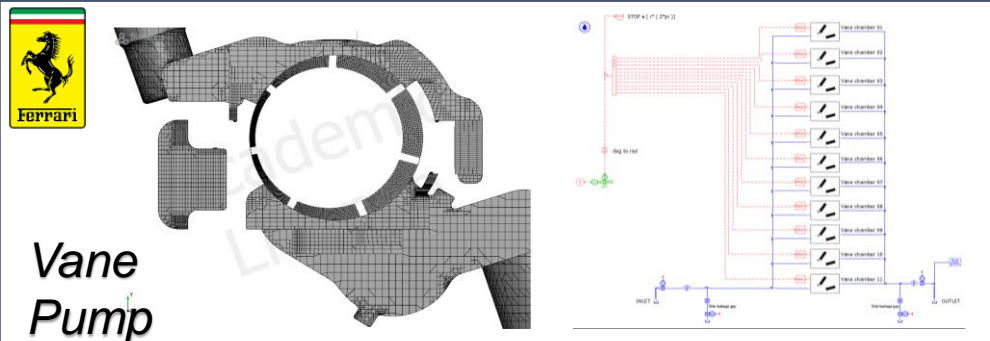
West Lafayette

June 4th, 2019

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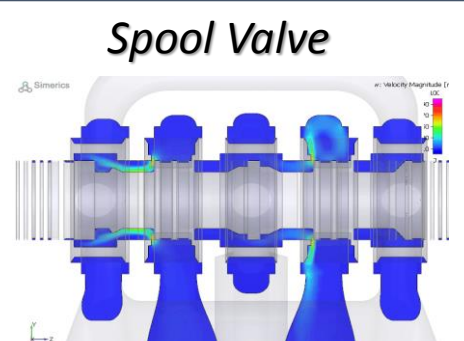
- **FPRG** – *University of Naples “Federico II”*
- Vane pumps:
 - Vane Dynamic
 - Chamber filling
- Piston pumps
 - Valve plate design model
 - Lumped parameter CV approach

Numerical modeling simulations: 1D and 3D CFD approaches

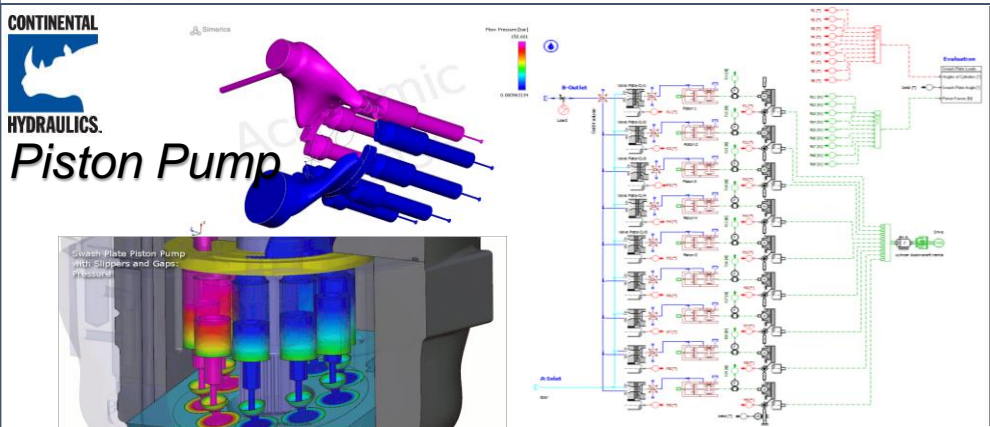


Ferrari

Vane Pump



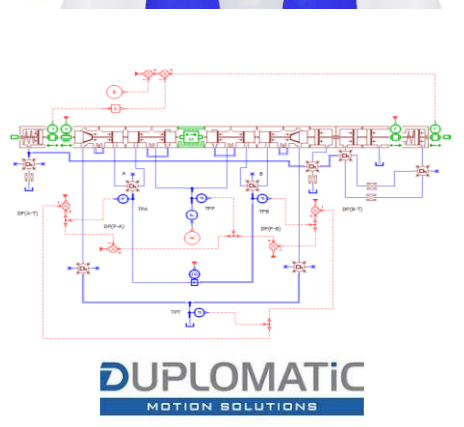
Pool Valve



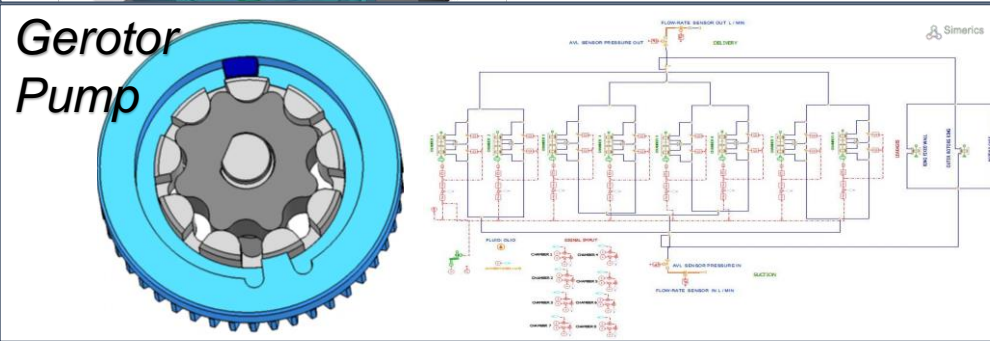
CONTINENTAL HYdraulics

Piston Pump

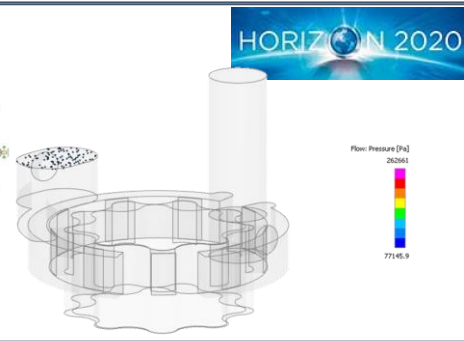
Swash-Plate Piston Pump with Slippers and Gaps



DUPLOMATIC MOTION SOLUTIONS



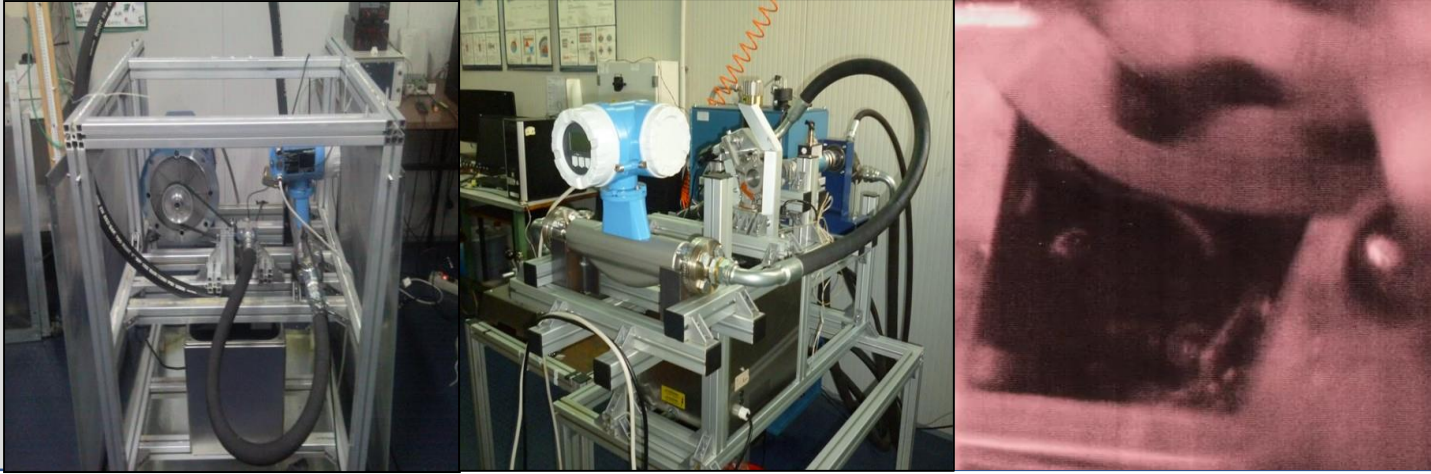
Gerotor Pump



HORIZON 2020

Pumps: experimental setup

*Lubrication
Pumps*

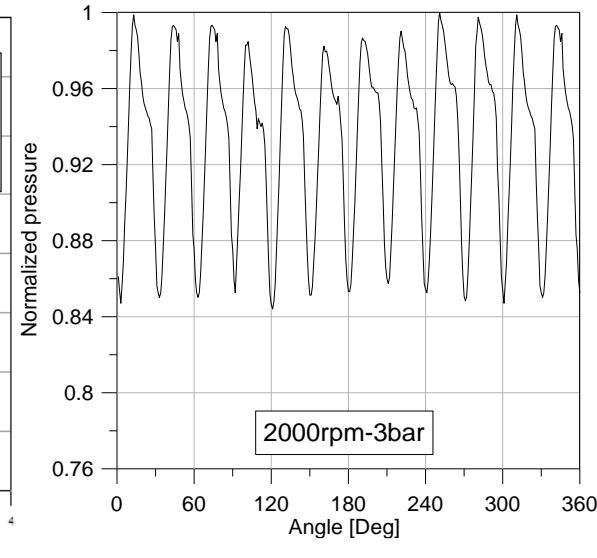
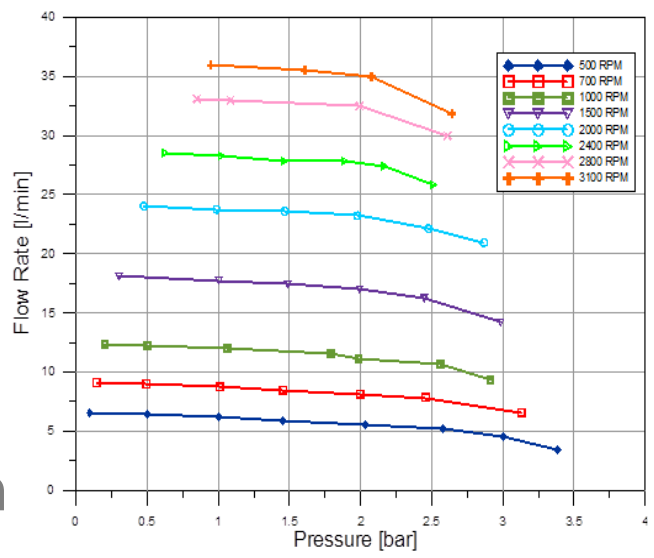


Pressure ripple

Flow-rate

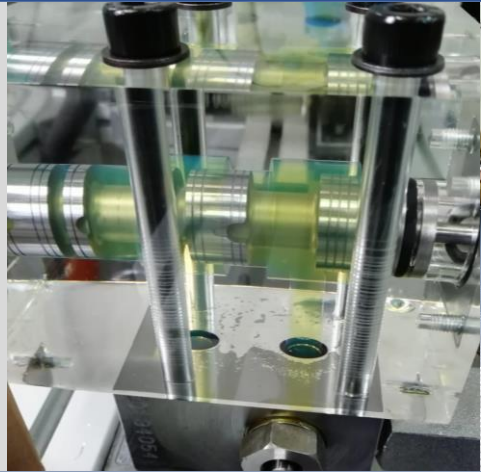
Torque

Cavitation phenomenon



Valves: experimental setup

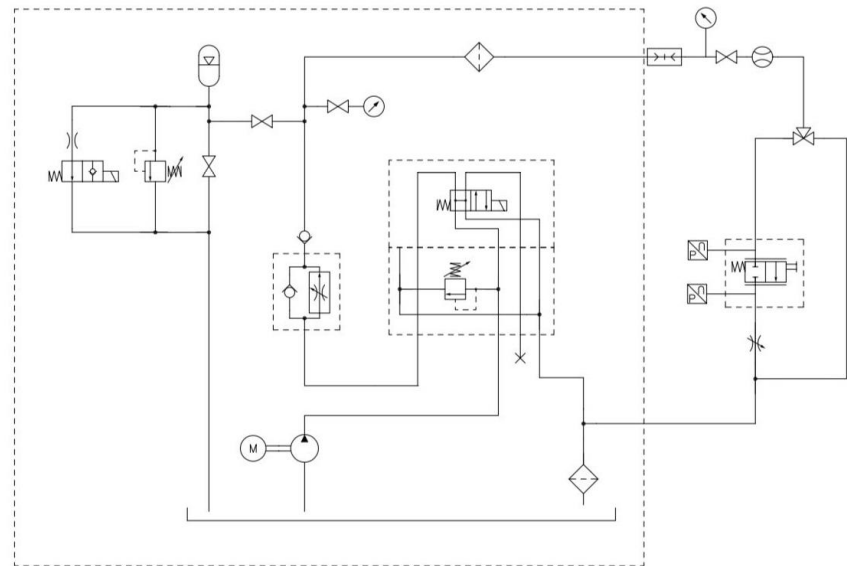
*Hydraulic
valves*



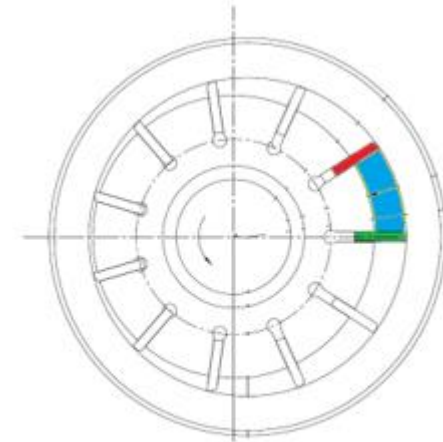
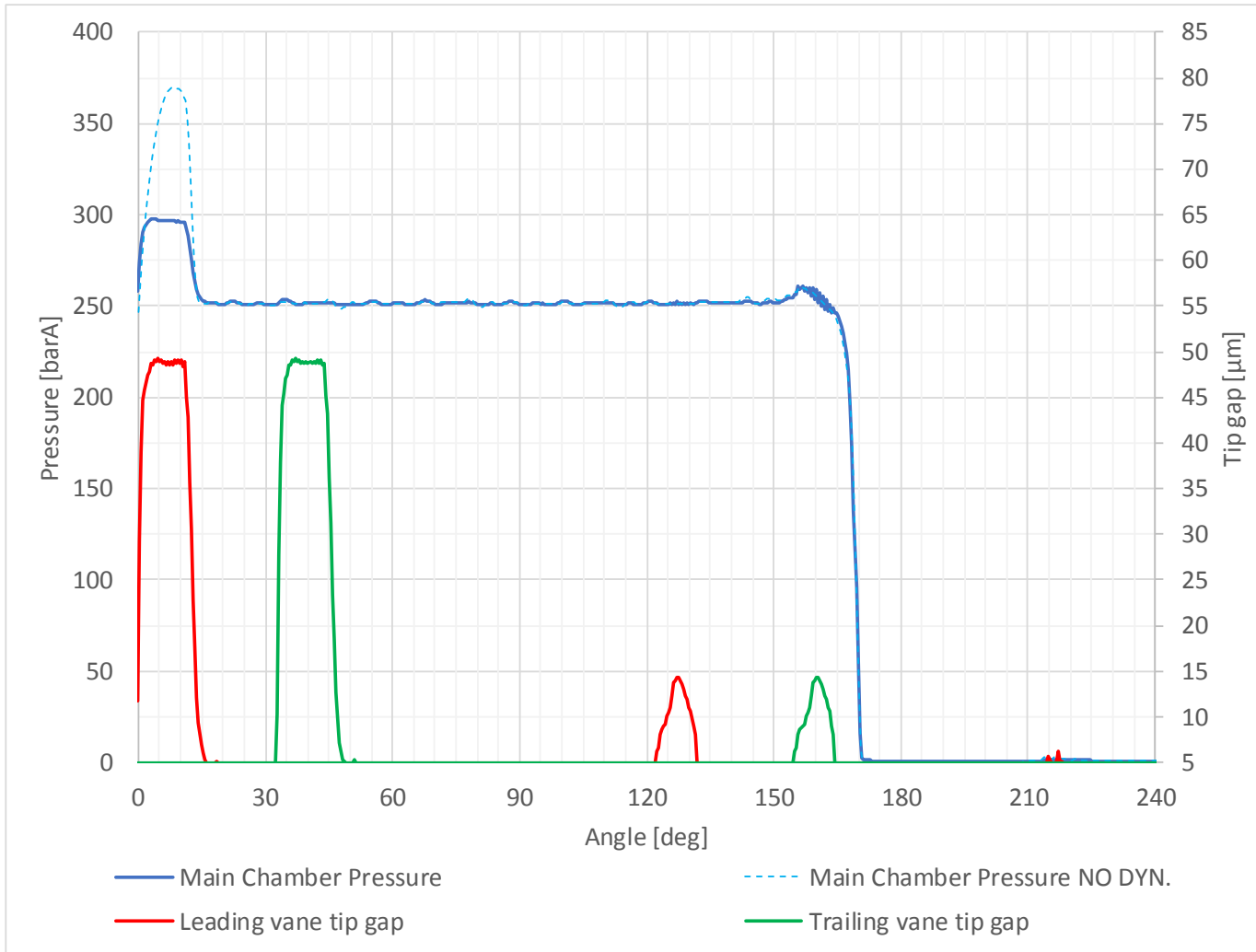
Pressure

Flow-rate

Cavitation phenomenon



Vane dynamic



Trailing Vane

Leading Vane

Main Chamber **MC**

Vane dynamic

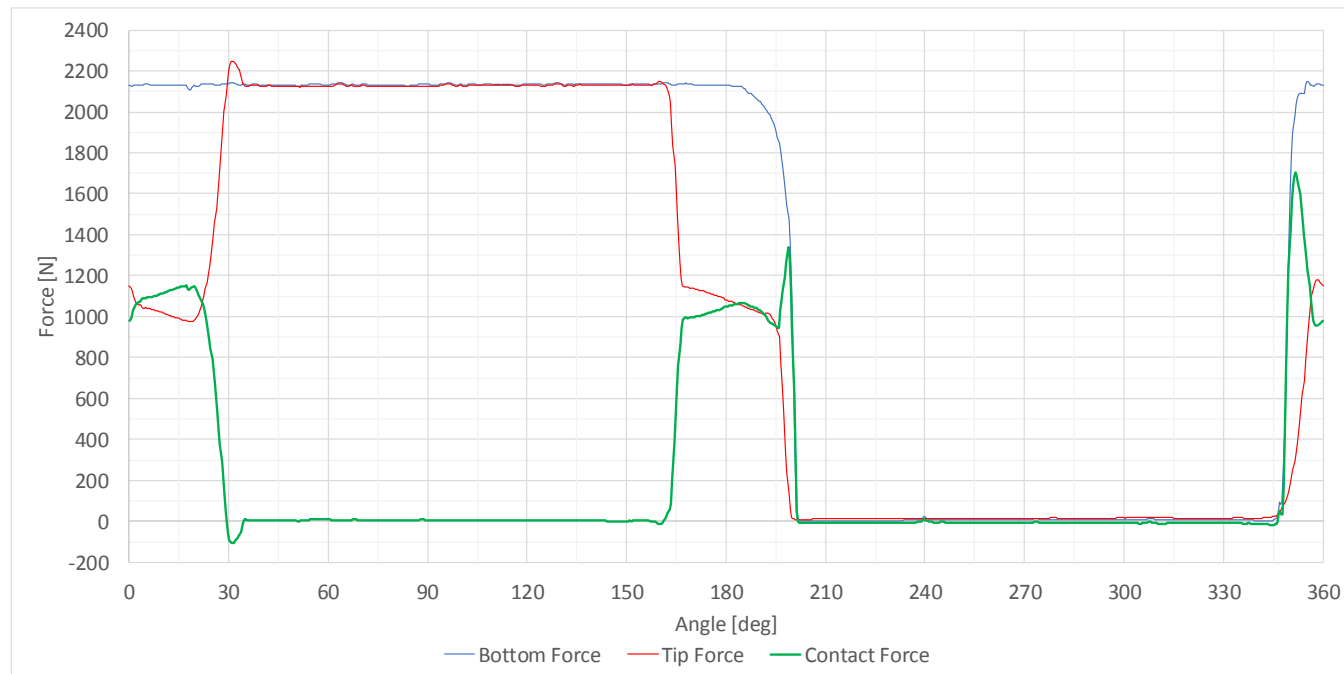
The contact force for each vane can be evaluated using the following equation:

$$F_t = F_p + F_c + \cancel{F_f}$$

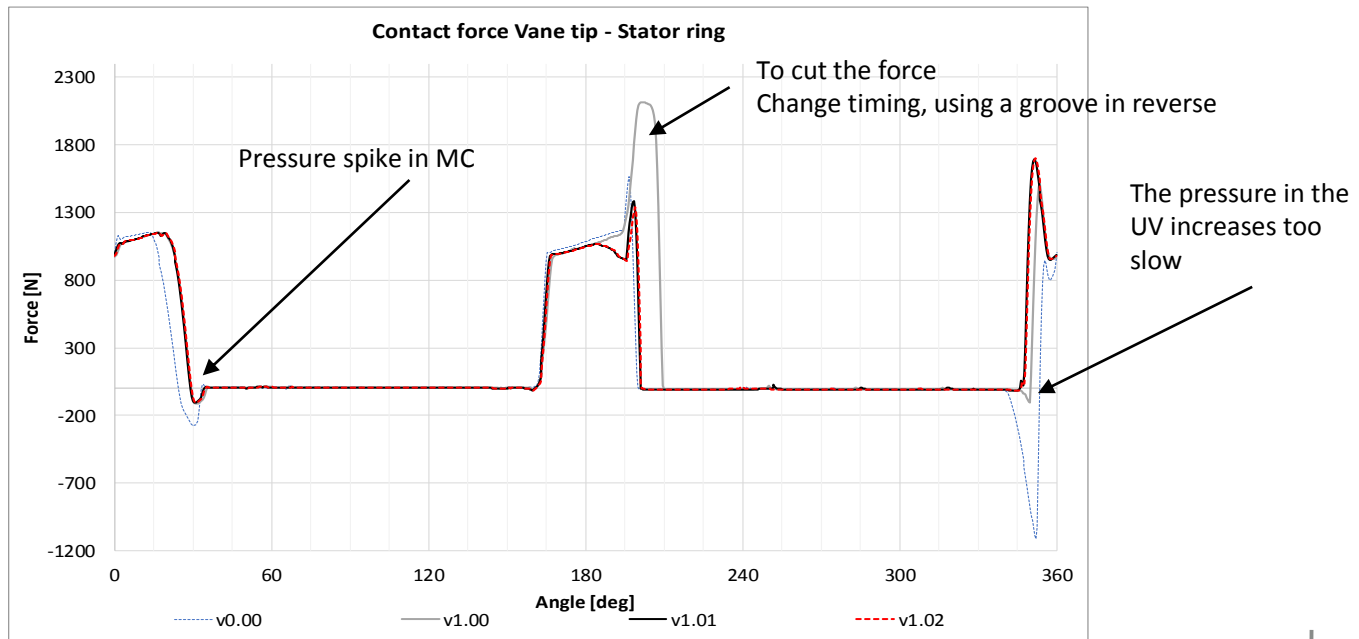
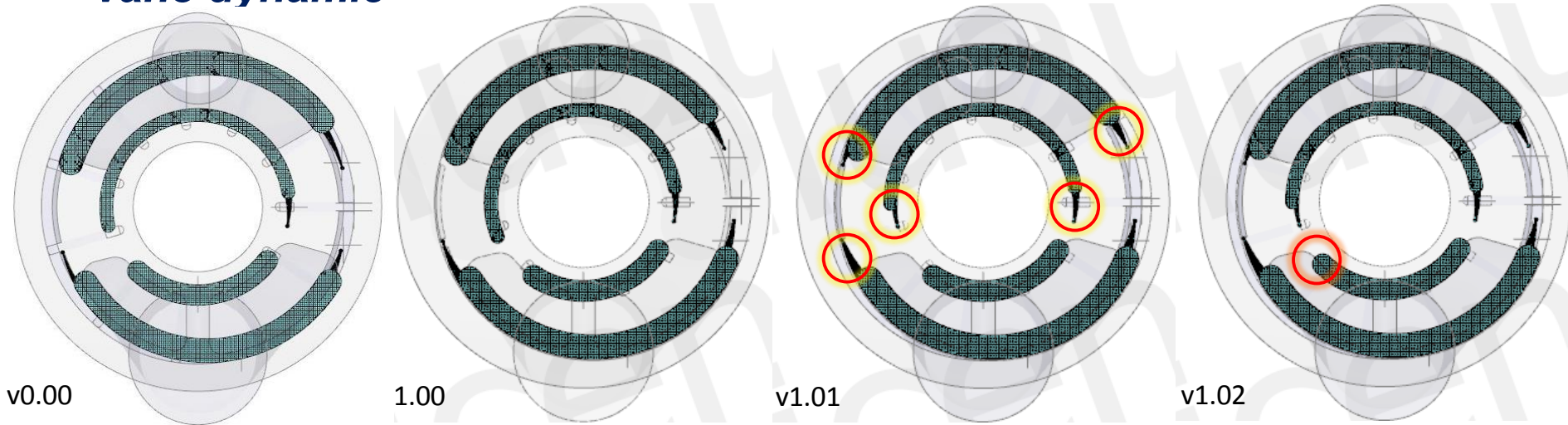
Where:

- $F_p = p_{r,uv} \cdot A_{uv} - p_{r,tv} \cdot A_{tv}$

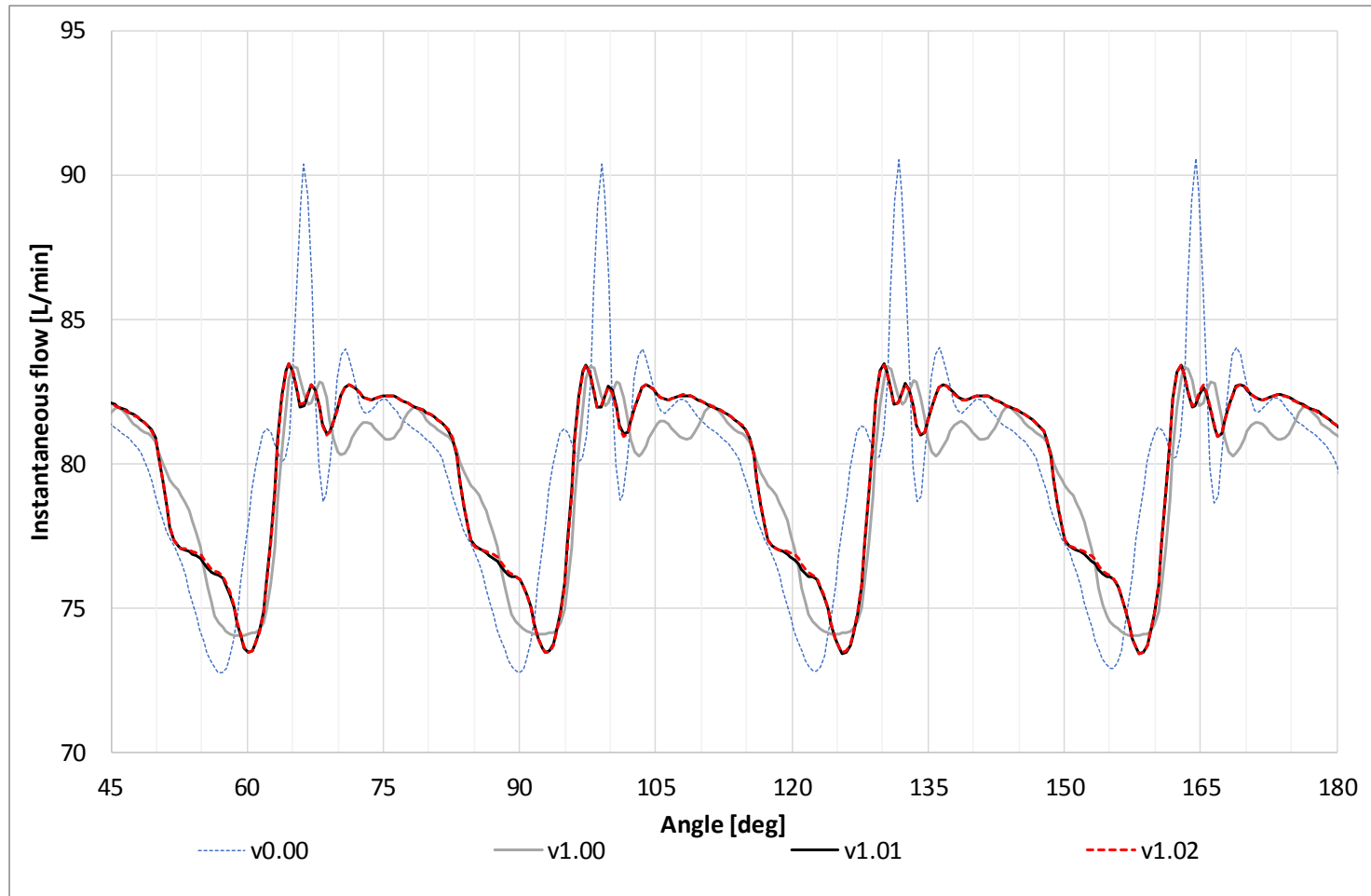
- $F_c = m_v \cdot \omega^2 \left(\rho - \frac{l_v}{2} \right)$ The vector ray ρ is: $\rho = \sqrt{e^2 + R_S^2 + eR_S \cos(\varphi + \delta)}$



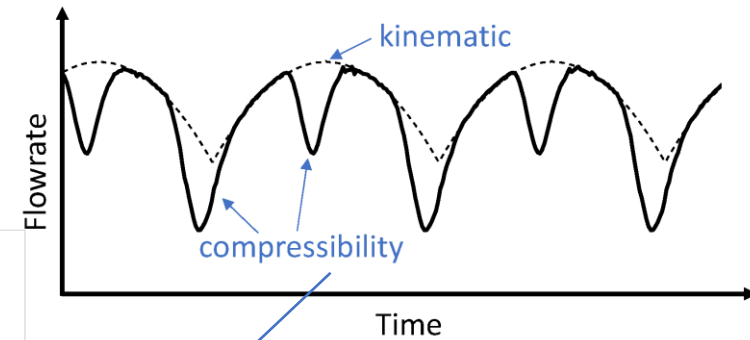
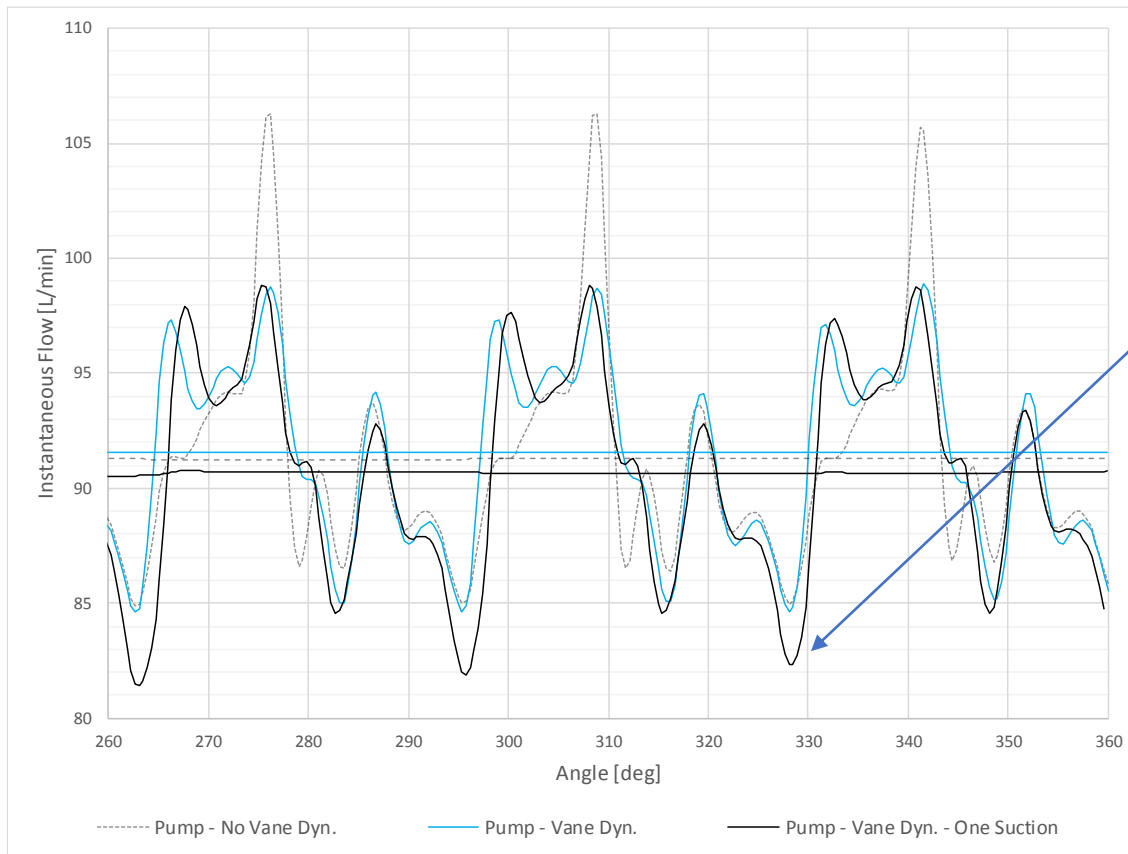
Vane dynamic



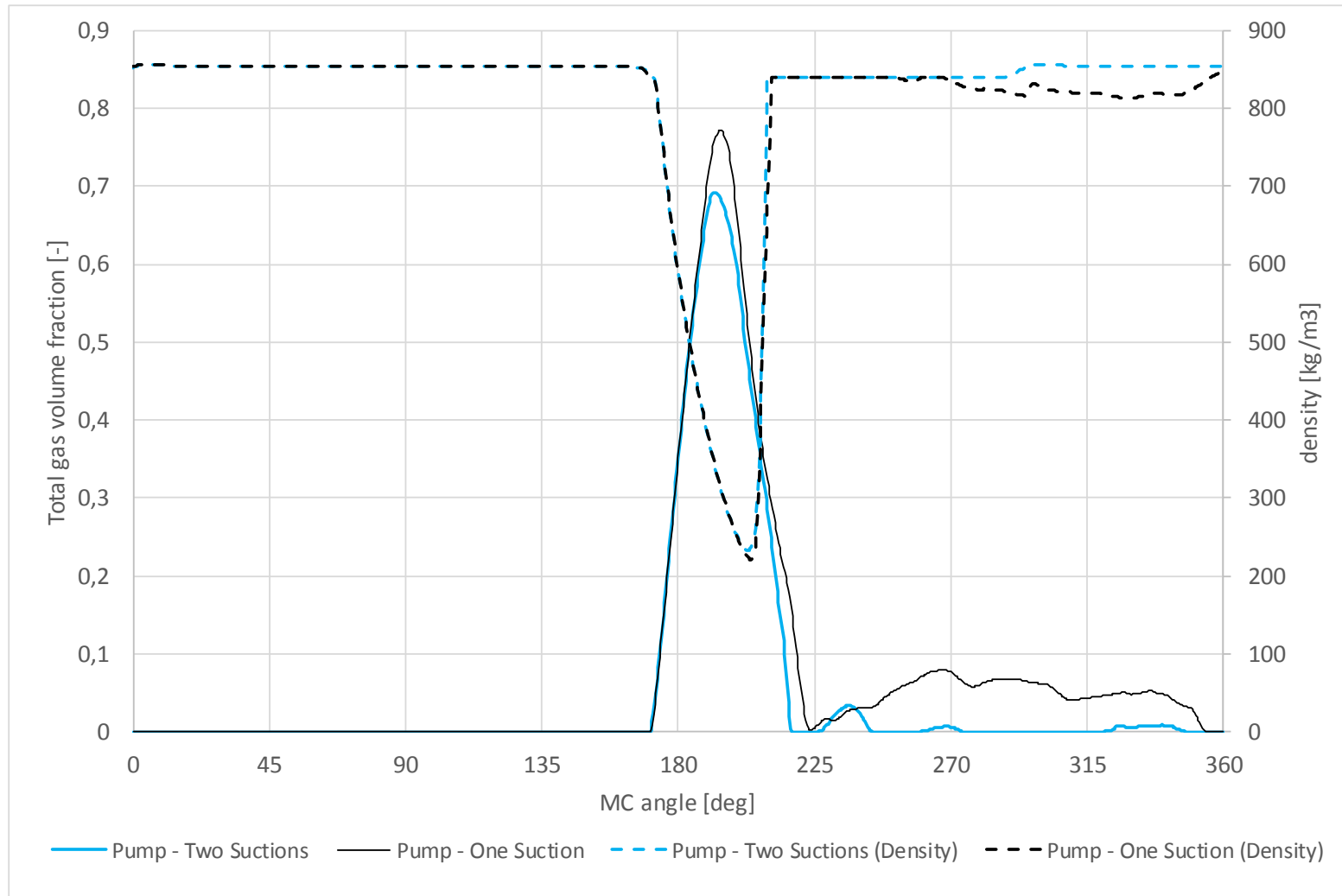
Vane dynamic



Chamber filling

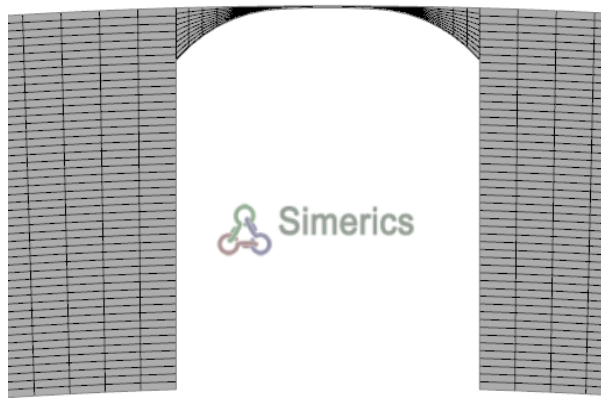


Chamber filling

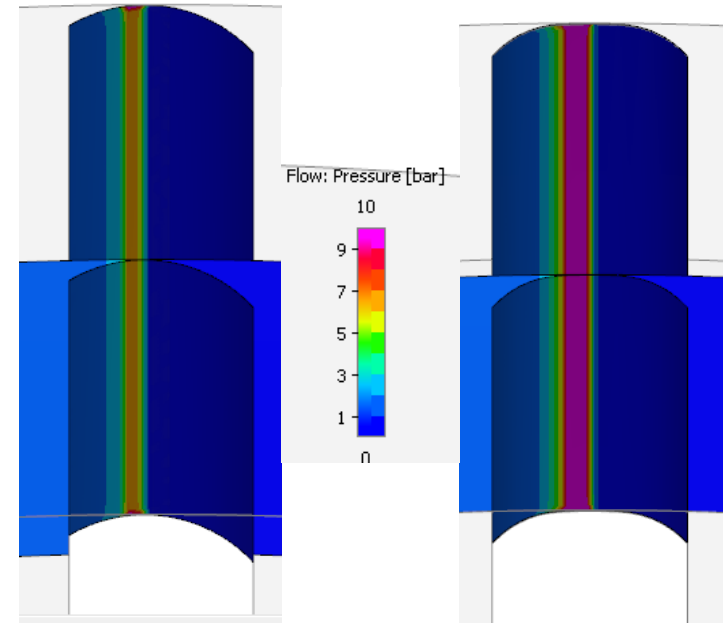


Vane Pumps: Future Works

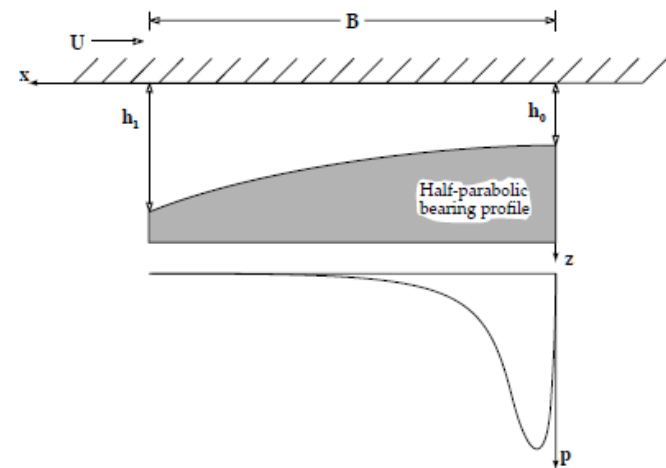
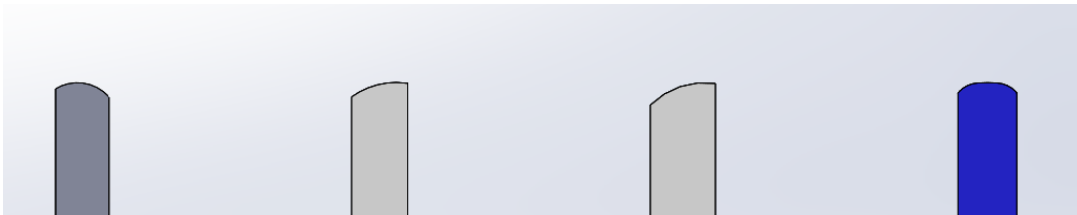
Vane tip profile



Hydrodynamic Load Capacity



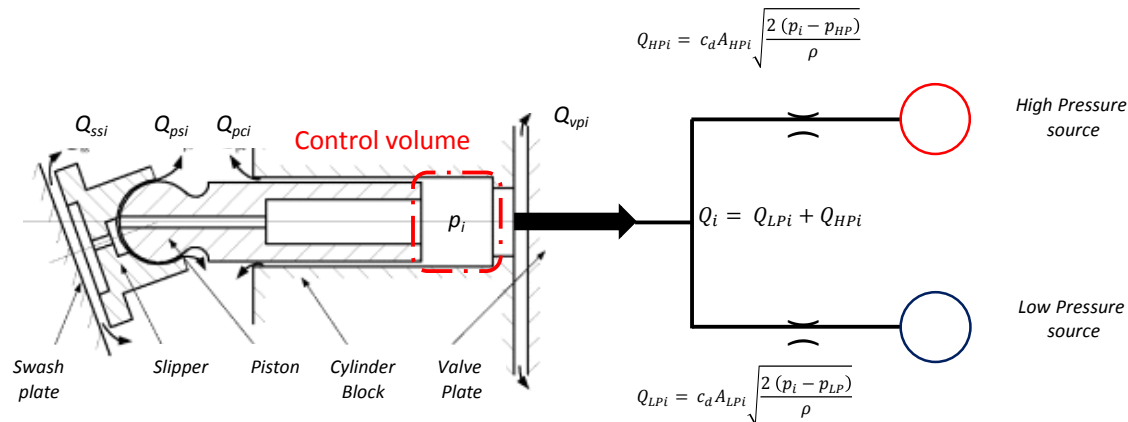
Different vane types:



Piston Pumps

Lumped parameter – Control volume approach

- Each control volume CV has homogeneous properties, such as pressure and temperature;
- CV is a capacitive element, it calculates the pressure as function of the net ingoing flow rate;
- CVs are connected by resistive components (variable orifices), calculating flow rate as function of the pressure drop.



$$\frac{dp_i}{dt} = \frac{\beta}{V_i} \left(Q_i - Q_{ssi} - Q_{psi} - Q_{pci} - Q_{vpi} - \frac{dV_i}{dt} \right)$$

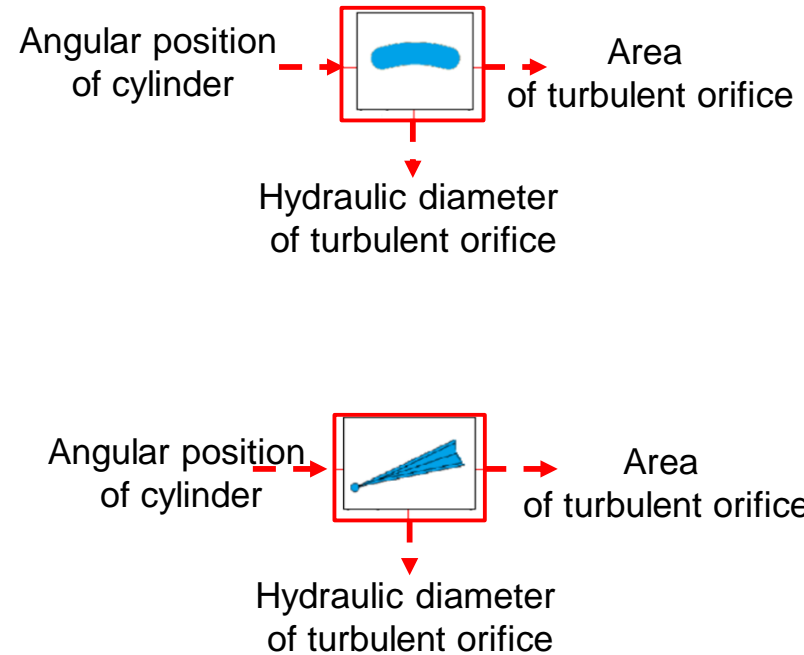
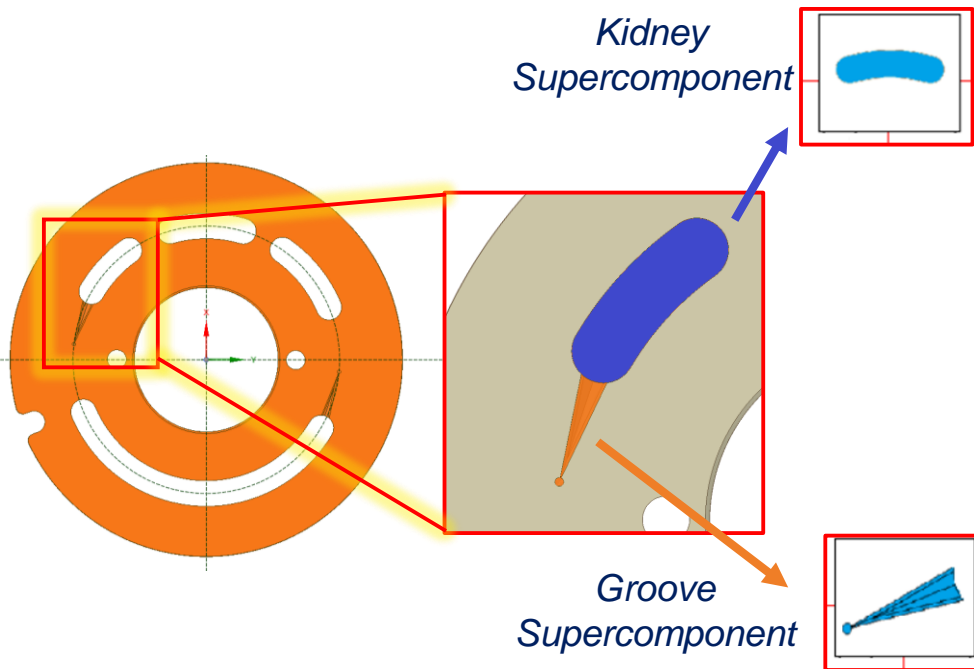
- The approach is consisting of two simulations:
 1. **Valve plate Design**, where the valve plate sectional areas are evaluated: the outputs of this model (tables) become inputs for the following model;
 2. **Vectorial piston pump**, modeled according to the CV approach

Piston Pumps

1 – Valve plate design model

The model has two main supercomponents:

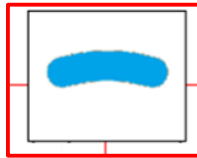
- **Groove SC**, to evaluate the slots;
- **Kidney SC**, to evaluate the kidneys opening areas.



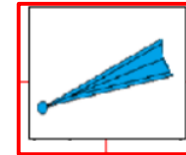
Piston Pumps

1 – Valve plate design model

Real Parameters:



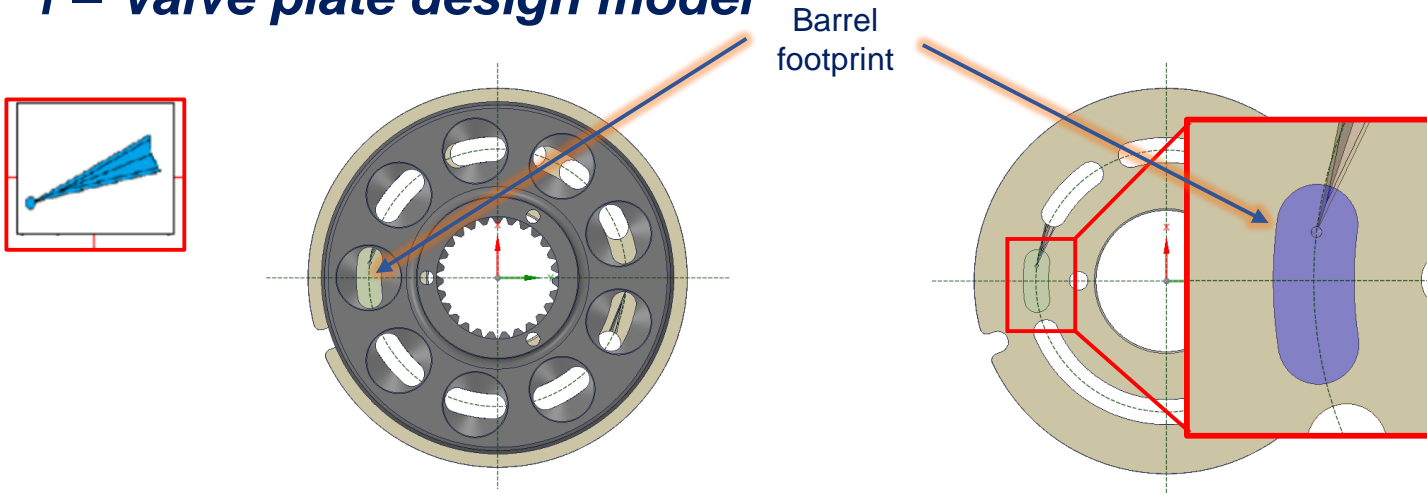
1. Pitch radius
2. Barrel footprint angular extension
3. Barrel slot radius
4. Kidney radius
5. Kidney opening angle
6. Kidney closing angle



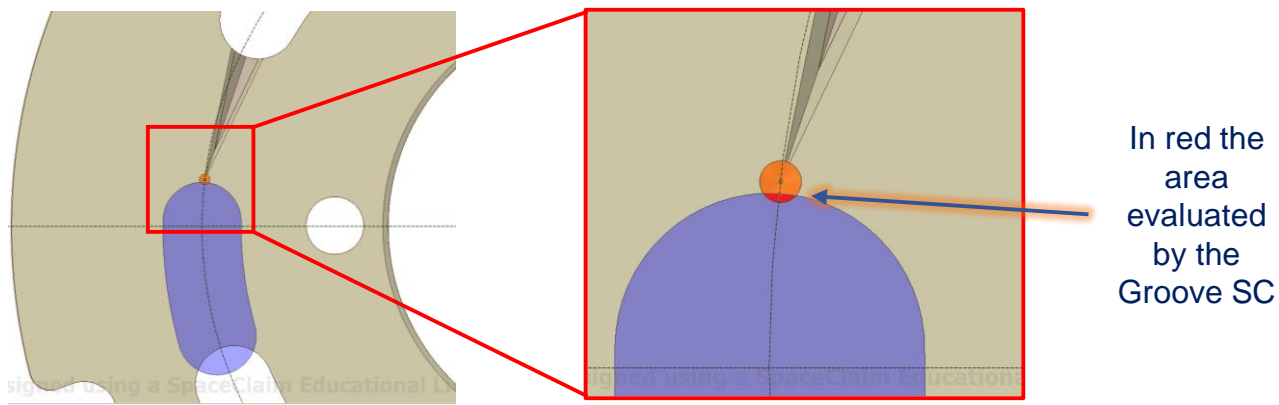
1. Pitch radius
2. Barrel footprint angular extension
3. Barrel slot radius
4. Hole radius
5. Hole opening angle
6. Hole closing angle
7. Slot starting height
8. Slot slope
9. Slot angle design (triangular slot)

Piston Pumps

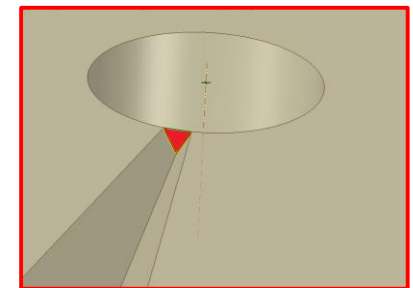
1 – Valve plate design model



Hole area:

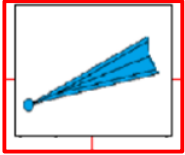


Limited by:

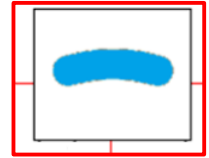
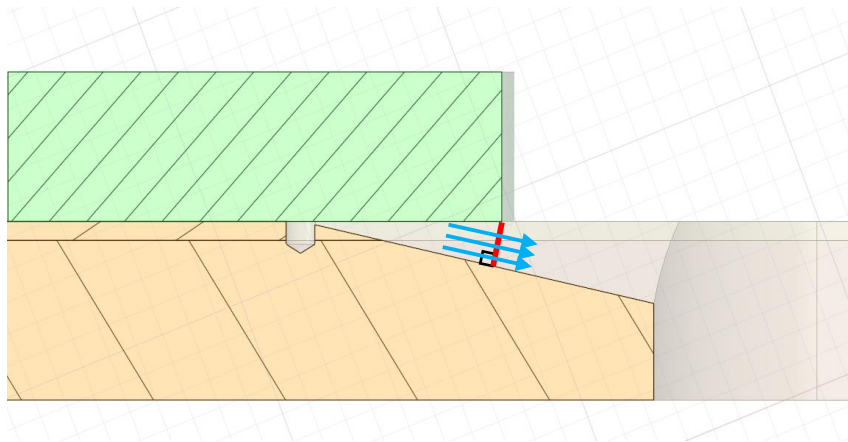


Piston Pumps

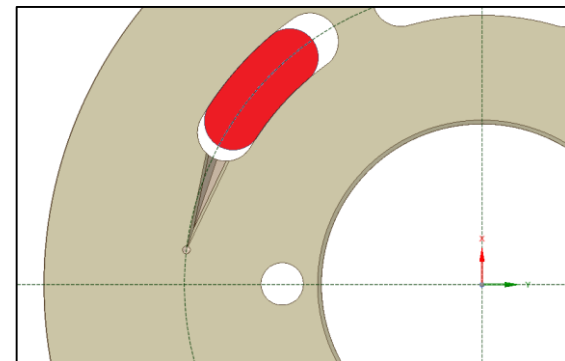
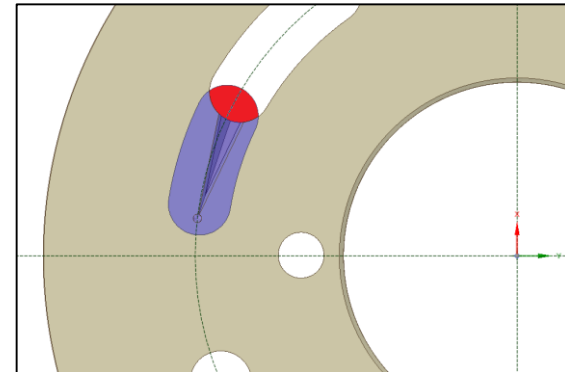
1 – Valve plate design model



Slot area: the slope angle is considered



Kidney area and its limit (barrel footprint area):

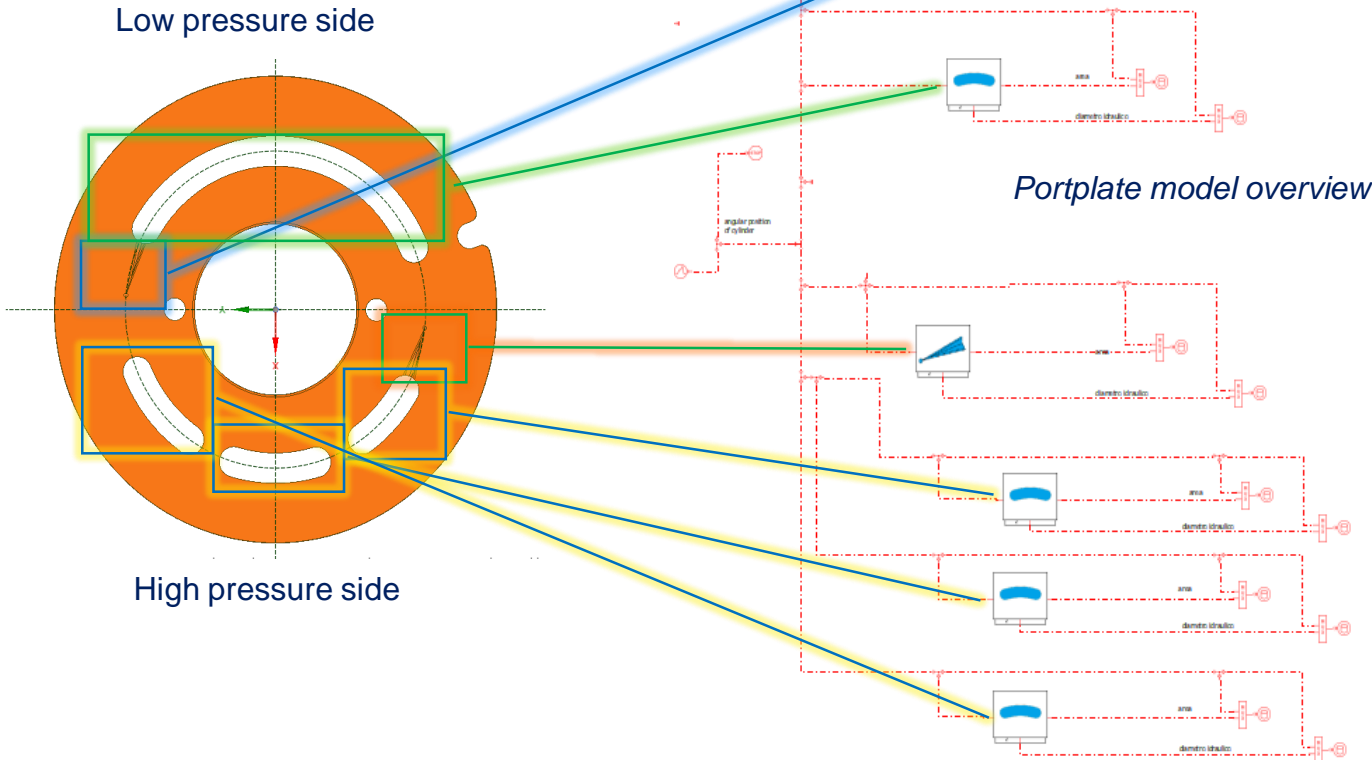


Piston Pumps

1 – Valve plate design model

Example of modeling:

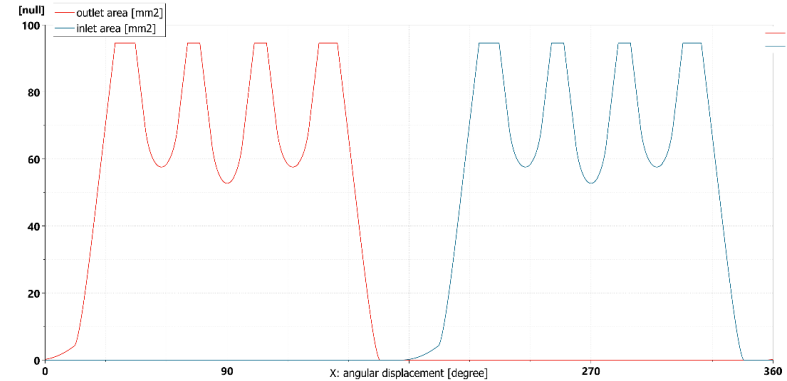
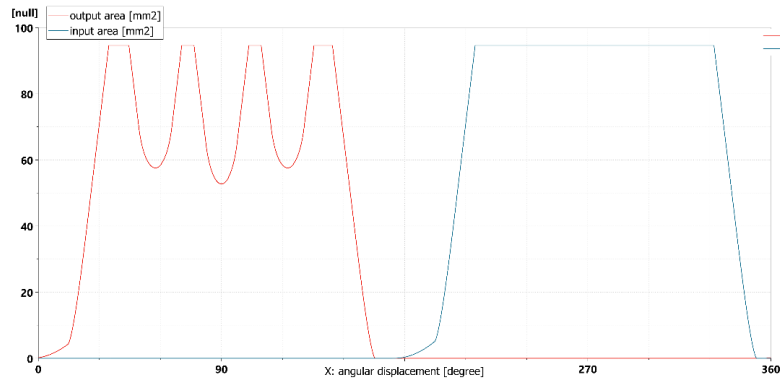
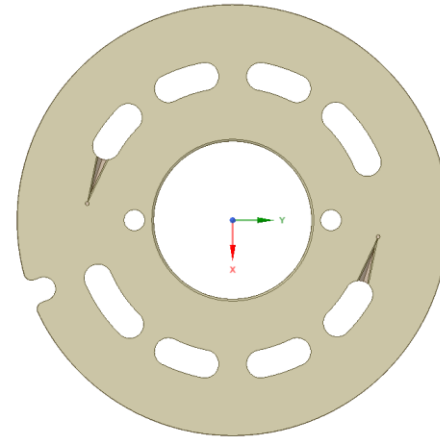
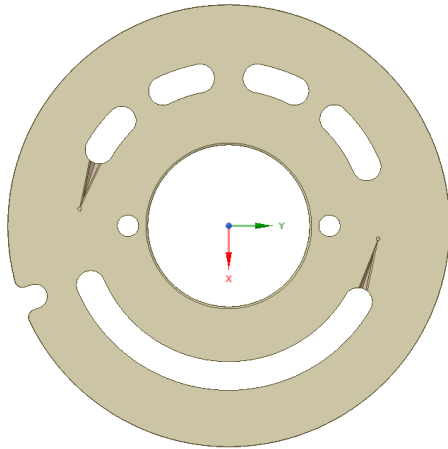
Portplate to model



Piston Pumps

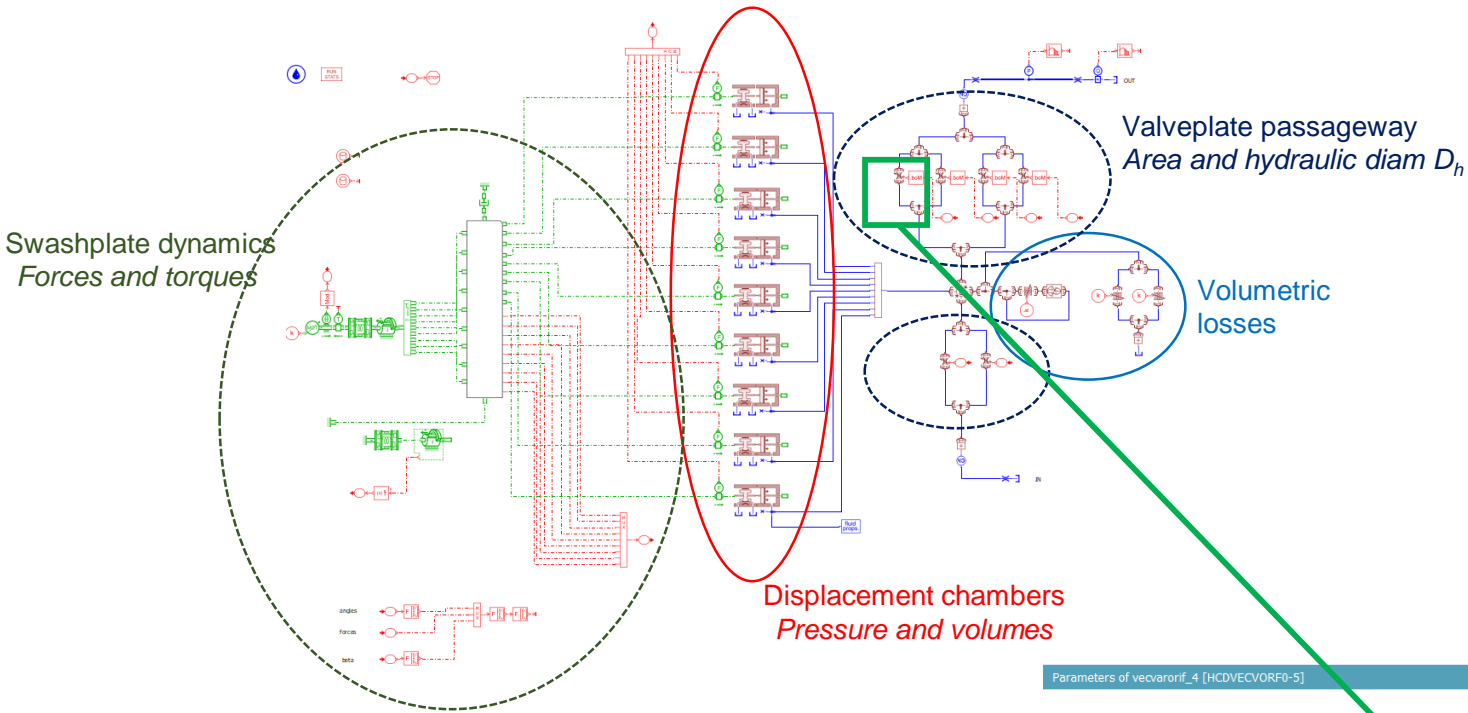
1 – Valve plate design model

Example of modeling:



Piston Pumps

2 – Lumped parameter CV approach



Parameters of vecvarorf_4 [HCDVECVORF0-5]

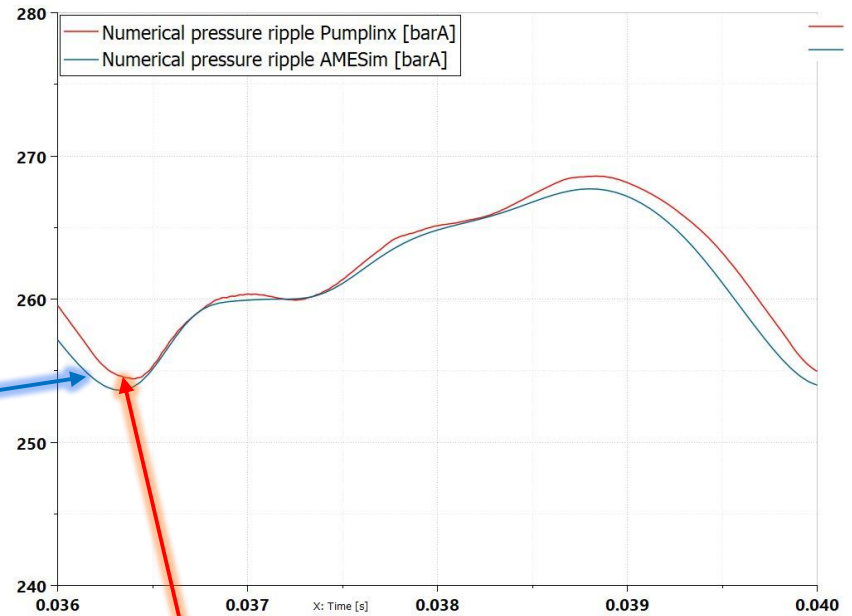
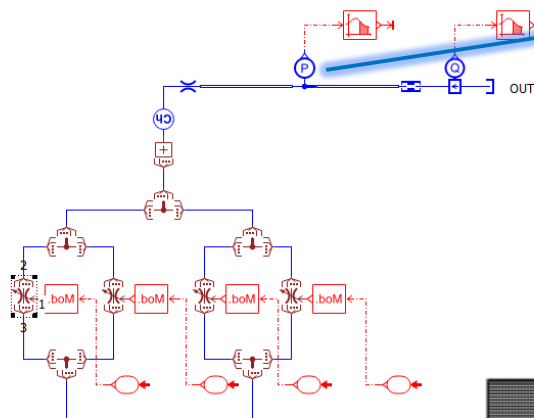
Title	Unit	Tags	Value	Name
index of hydraulic fluid			0	indexf
vector size			9	z
number of signal inputs			1	nbinp
geometry of orifices				regular mode
maximum flow coefficient Cq	null		0.7	cqmax
critical flow number (laminar/turbulent)	null		100	lamc
periodicity	degree		360	period
filename or expression for area[mm**2] = f(theta)				Outlet_groove_AREA.data area1
filename or expression for hydraulic diameter(mm) = f(theta)				Outlet_groove_HYD_DIAM.data hd1
▶ table settings				

Piston Pumps

Approach validation

The approach is validated comparing the results with a CFD simulation:

Here is showed a comparison on a pressure ripple:



Pressure monitoring point (CFD3D)

Thank you for your attention

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